



Heat load and Duct Design of Economical Air Duct - A Case Study at Stress Analysis Lab in Anjuman College of Engineering and Technology Nagpur

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1.1 INTRODUCTION

In today's circumstances using an air conditioner is not an optimal option as it consumes lot of electricity and we get a hefty electricity bill. For conservation of energy and to save money on electric bills opting for an air cooler ducting (HVAC) is the best option under such circumstances. As the college's is present in a hot and less humid region it justifies the use of air cooler ducting more than an air conditioner. The air cooler duct is designed on calculations referring from the ISHRAE books- Load Calculation Applications Manual and duct system and design guide.

2.1 HEAT LOAD CACULATION

I. Two types of wall area

1. Gross wall
2. Net wall area

$$Q = U \times A \times T$$

Where,

U= Overall heat Coefficient.

$$U = 1 / \text{sumR}$$

sumR = (Resistance of susbtance)

$$A = 34.6 \times 16.2$$

$$= 560\text{sq.ft}$$

$$\Delta t = 106 - 76 = 30^\circ\text{F}$$

1. Gross wall area

$$Q = U \times A \times T$$

$$A = 179 \text{ sq.ft}$$

$$\Delta t = 30^\circ\text{F}$$

ISHRAE STANDARD VALUES

$$\text{SumR} = 0.25 + \frac{1}{2} \times (0.12) + (8 \times 0.2) + \frac{1}{2} \times 0.12 + 0.25$$

$$\text{SumR} = 2.2$$

$$U = \frac{1}{\text{sumR}} = \frac{1}{2.2} = 0.45 \text{ BTU/hr. F.Ft}$$

$$\therefore Q = UA\Delta t$$

$$Q = 0.45 \times 138 \times 30$$

$$Q = 1863 \text{ BTU/hr (1w = 3.14 BTU/hr)}$$

$$Q = 593.31 \text{ W}$$

$$Q = 0.593 \text{ kW}$$

2. Roof area

$$Q = U \times A \times \Delta T$$

$$U = \frac{1}{\text{SumR}}$$

$$U = \frac{0}{A} \times \frac{1}{2} \times (C.P) + 8 \text{ inch brick} + \frac{1}{2} \times (C.P) + I.A$$

$$U = 0.25 \times \frac{1}{2} + 0.12 + 8 \times 0.08 + \frac{1}{2} \times 0.12 + 0.65$$

$$\text{SumR} = 1.72$$

TAKING STANDARD ISHRAE VALUES

$$U = \frac{1}{\text{SumR}} = \frac{1}{1.72} = 0.58 \text{ BTU/hr F.ft}$$

$$A = 30 \times 14$$

$$A = 420 \text{ sq.ft}$$

$$\Delta T = 106 - 76 = 30 \text{ F}$$

$$Q = U \times A \times T$$

$$Q = 0.58 \times 420 \times 30$$

$$Q = 7308 \text{ BTU/hr F.ft}$$

$$Q = 2327.38 \text{ W}$$

$$Q = 2.32 \text{ kW}$$

Internal heat load

1. People heat load

2. Lighting heat load

3. Electrical heat load

1. People heat load

$$Q_s = \frac{\text{Sensible heat}}{\text{people} \times \text{no. of people}}$$

$$Q_l = \frac{\text{Latent heat}}{\text{people} \times \text{no. of people}}$$

Ex. Lab room = 420 sq.ft = 3 people

Sensible heat (from the carts of ISHRAE)

For seated light office work = 245

Latent heat (from the carts of ISHRAE) = 155

$$Q_s = 245 \times 3$$

$$Q_s = 0.735 \text{ BTU/hr} = 0.23 \text{ kW}$$

$$Q_l = 0.465 \text{ BTU/hr} = 0.14 \text{ kW}$$

2. Lightning heat load

$$Q = \text{watt/sq.ft} \times 3.4$$

For office = 1.1 watt/sq.ft

$$Q = 1.1 \times 420 \text{ sq.ft} \times 3.4$$

$$Q = 1570.8 \text{ BTU/hr}$$

$$Q = 0.50 \text{ kW}$$

3. Electrical heat load

$$Q = \text{total equipment load (watts)} \times 3.4$$

$$Q = 440 \times 3.4$$

$$Q = 1,496 \text{ BTU/hr}$$

$$Q = 438.70 \text{ w}$$

$$Q = 0.45 \text{ kw}$$

TOATL HEAT =4.5KW

SINCE 3.5 KW = 1 TON OF AIR CONDITIONING

THEREFORE 4.5 KW = 1.3 TON \approx **1.5 TON OF AIR CONDITIONING**

A 1.5 ton air conditioner (AC) typically uses 1.2–1.5 kilowatt (KW) of electricity per hour, depending on the temperature. For example, a 5-star AC uses about 1.5 KW per hour, while a 3-star AC uses 1.6 KW per hour. If the AC is used for 7 hours a day, it might consume around 360–480 units of electricity in a month

According to Maharashtra (India) price of electricity per units = 7 rupees per unit consumed

Assuming 480 units consumed per month = $7 \times 480 =$ **3,360 rupees per month**

Therefore going for a cooler ducting is an optimal solution to reduce the power consumption

2.2 DUCT DESIGN BY EQUAL FRACTION METHOD

$$M^{\circ} = Q / c_p \times \Delta T$$

Where,

(M) mass flow rate = kg/s

Q(kW) heat load = 4.5 kW

Cp = Specific heat capacity (kJ/kgK)

= Cp1.026 (kJ/kgK) (standard value from ISHRAE)

ΔT = temperature difference

ΔT should be less than 10 °C

$\Delta T = 8^{\circ}\text{C}$

$$M^{\circ} = 4.5 / 1.026 \times 8$$

$$M^{\circ} = \mathbf{0.6 \text{ kg/s}}$$

$$M = \text{kW} / (\text{kJ/kgK} \cdot 8\text{K})$$

Density of Air = 1.2 kg/m³

Specific volume = Density-1.2-1 = 0.833 m³/kg

Formula: -

$$V^{\circ} = M^{\circ} \times v$$

V^o = volume flow rate (m³/s)

M^o = mass flow rate (kg/s)

v = specific volume (m³/kg)

$$v = 0.6 \times 0.833 V = 0.49 \text{ m}^3/\text{s} \text{ (1cubic meter/second} = 2118.8 \text{ cfm)}$$

So,

$$V \approx 0.49 \times 2118.8$$

$$V \approx 1038.21 \text{ cfm}$$

68°F Air STP

Fluid density: 0.075 lb/ft³
 Fluid viscosity: 0.0432 lb/ft-h
 Specific Heat: 0.24 Btu/lb°F
 Energy factor: 1.08 Btu/h°F-cfm

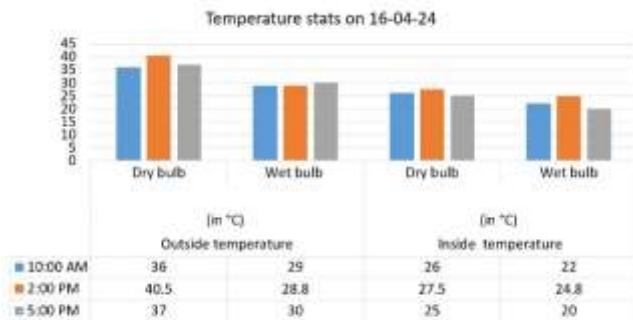
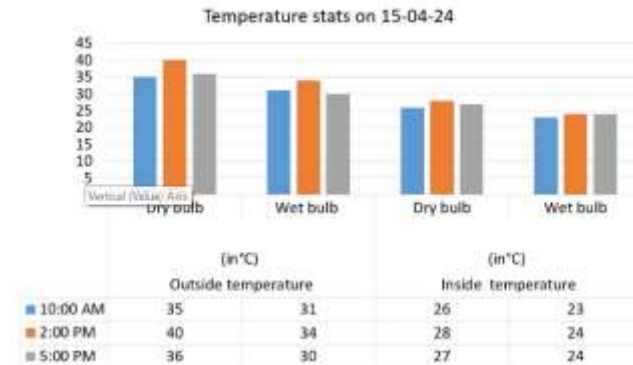
Flow rate: 1038.21 cfm
 Head loss: 0.174 in.WC/100 ft
 Velocity: 1200 fpm
 Equivalent diameter: 12.6 in

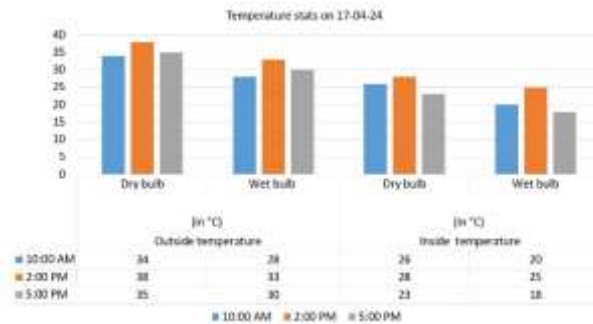
Duct size: 21 in X 23 in

Equivalent Diameter: 24.02 in
 Flow Area: 3.1465 ft²
 Fluid velocity: 330.0 ft/min
 Reynolds Number: 68,794
 Friction factor: 0.02089
 Velocity Pressure: 0.0068 in.WC
 Head Loss: 0.007 in.WC/100 ft

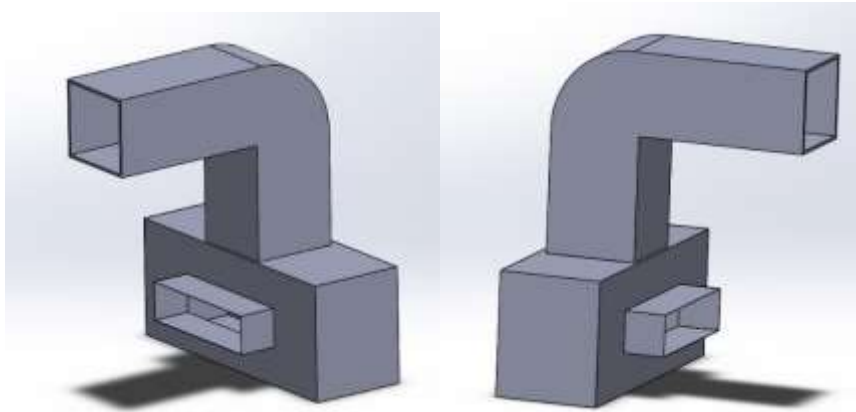
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3.1 EXPERIMENTATION





4.1 CAD MODEL OF ROOM AND 3D CAD OF DUCT



4.1 Conclusion

We were successful installing and operating our project with efficient utilization of resources. Hence we were able to provide ambient temperature, which was comfortable for teachers and students.

By taking in considering all the heat load factors while calculation of size of duct, we meet our goal to provide the necessary comfortable conditions to the students and teachers in Stress analysis lab

5.1 References

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